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ENERGY

Parameterized Friction Modeling With Optimized User Constructs

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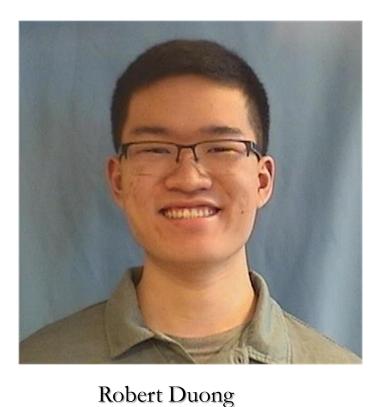
Sandia Vationa

horatories





Our Team 2









Chris Jawetz







TEXAS TECH UNIVERSITY. Georgia Institute of Technology.

³ Problem

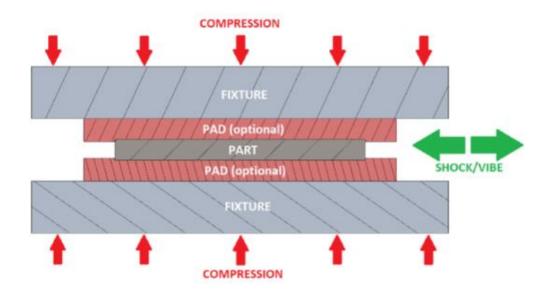
Many systems hold components in place

through clamping

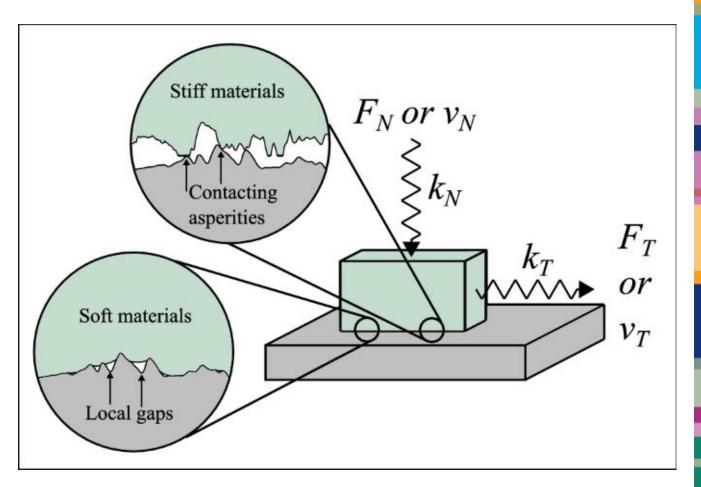
Friction is notoriously difficult to model

>Determining required clamping often takes

many cycles of destructive testing



- Compliant Friction Modeling
 - Micro scale surface features change friction coefficients
 - True surface area increases as compression increases
 - Compliant materials deform and "fill" voids from surface features



>FEA required for simulation

[1]

⁵ Our Project

Model

Build a 3D model in SolidWorks

> Mesh using CUBIT

Simulate in SIERRA under different conditions Build design advisor ≻ Import user conditions

Determine successes and failures

Find regions of success, failure and uncertainty Test and verify > Test at simulated points

Test at high preloads based on prediction function

Input test data into design advisor for comparison



Simulation & Modeling



Simulation Parameters

Efficiency of simulations were paramount

>Pad deformation is critical to the validity of the simulations

>Hundreds of simulations required

>Sweep over geometries, pad materials, and compressions

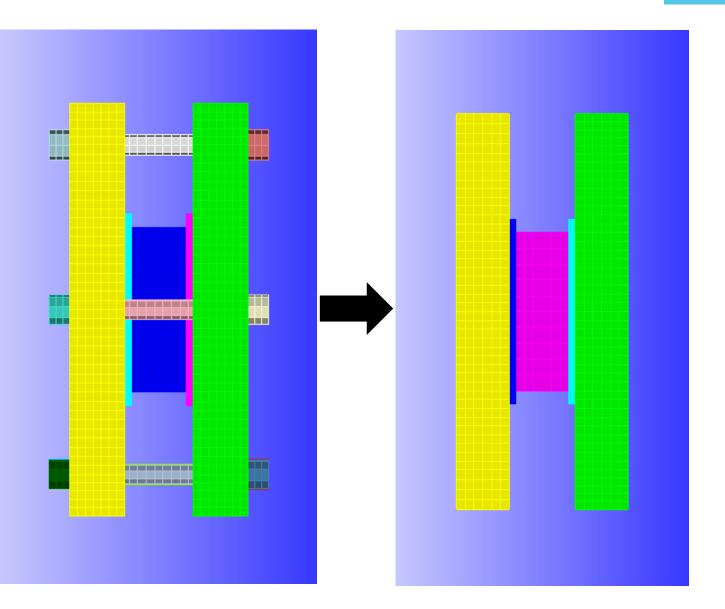
Automated through Python scripts

⁸ Simplified Geometry

Model on right was used for bulk simulations

>Verified against the exact model for stress distribution

>Allowed for 3x timestep size



Simulation Steps

Apply compression

- Uniform force across the top plate
- Force increases with cycloidal ramp
- Time to settle

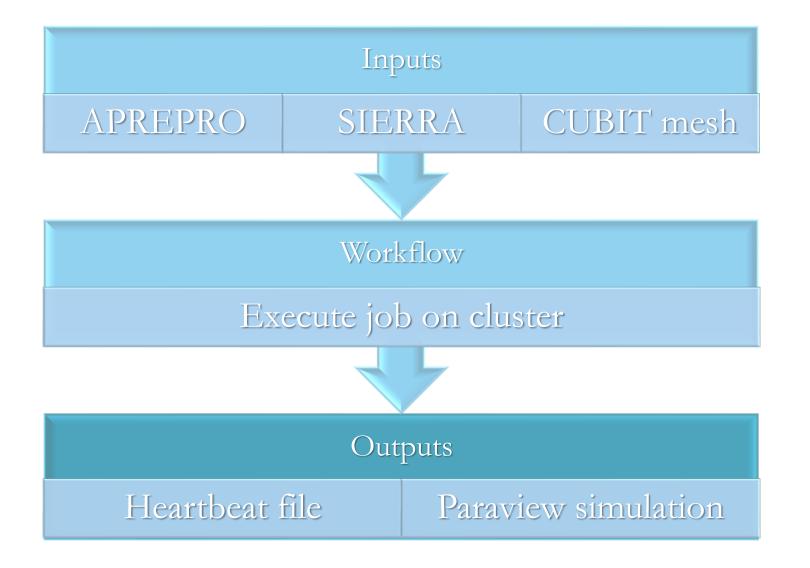
Apply shock

- 2ms Haversine pulse
- Allow it to dissipate

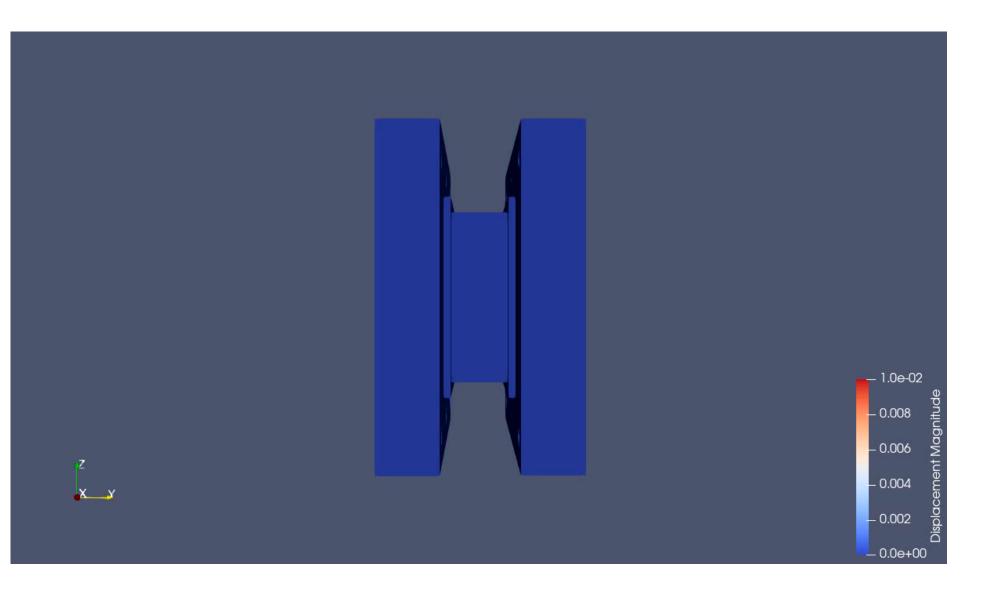
Output data

- Position data
- Acceleration data
- Element death

¹⁰ Simulation Workflow



¹¹ Simulation Results





Design Advisor



¹³ Design Advisor Simplified Overview

Inputs

- Simulated data
- User input data
- User fail criteria
- Requested output graphs

Data processing

- Import principal simulation data
- Determine maximal successes & minimal fails from failure criteria

Optional data processing

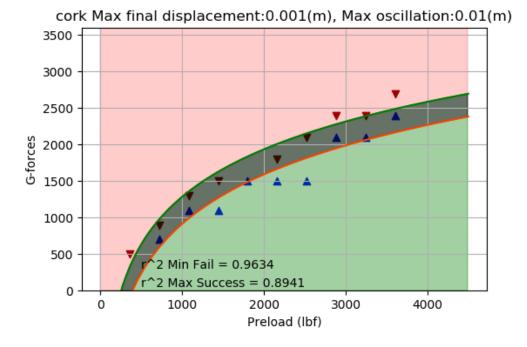
- Check additional requested graphs
- Import secondary simulation data
- Apply failure criteria

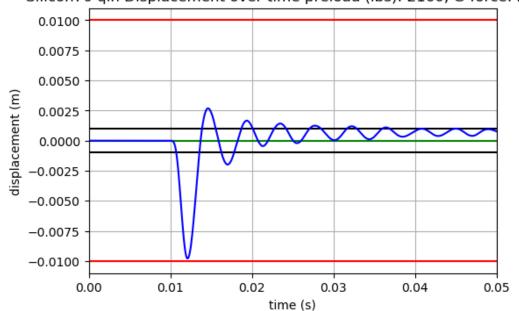
Outputs

• Plot best fit success & failure functions

- Plot failure criteria against displacement
- Save all figures to host computer

¹⁴ Design Advisor Outputs





Silicon70 qin Displacement over time preload (lbs): 2160, G-force: 2100

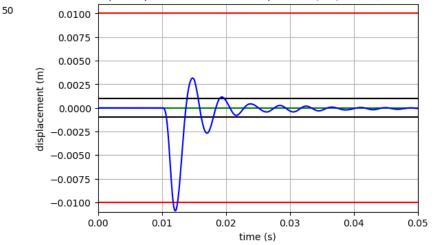
¹⁵ Displacement Over Time



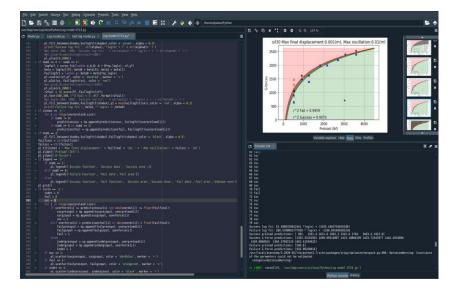




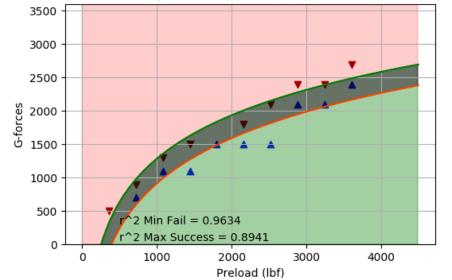
Silicon70 gin Displacement over time preload (lbs): 2520, G-force: 2400

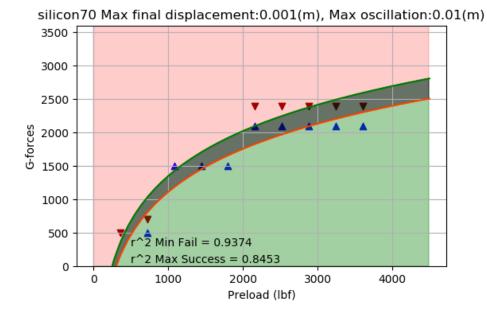


¹⁶ Design Advisor

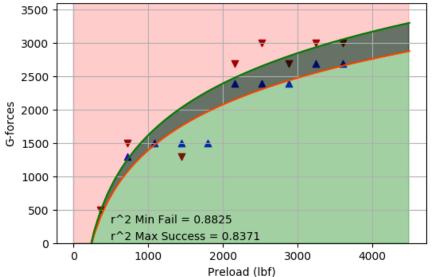


cork Max final displacement:0.001(m), Max oscillation:0.01(m)





silicon30 Max final displacement:0.001(m), Max oscillation:0.01(m)

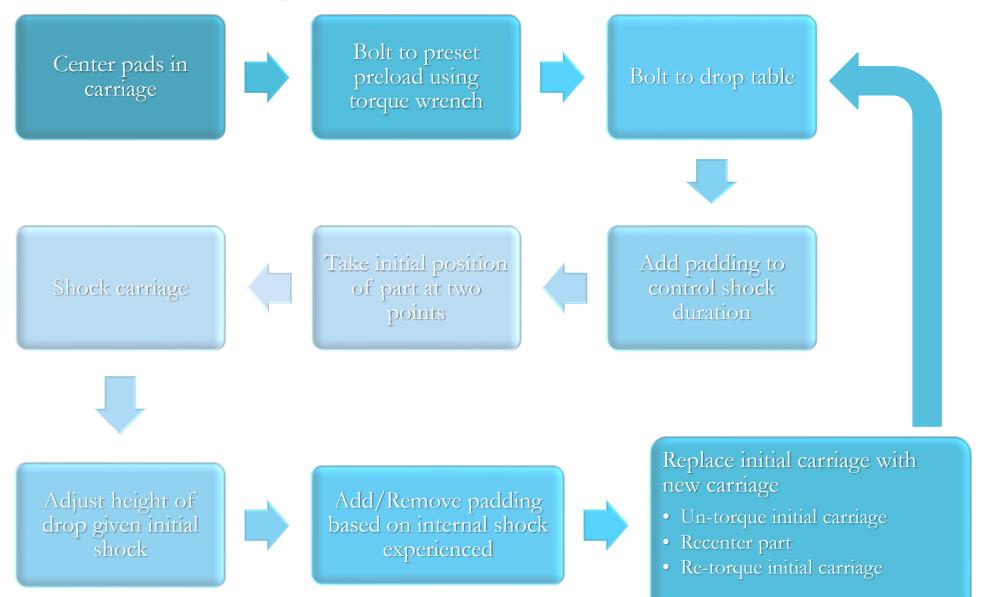




Verification & Testing



¹⁸ Test Methodology

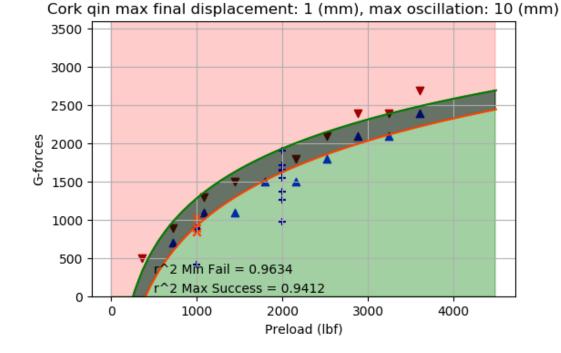


¹⁹ Data Collection

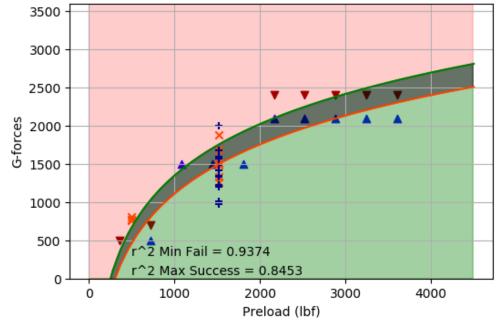
- ➢High speed video
 - Displacement over time
 - ➢Pad deformation and slip
- Accelerometers
 - Part kinematics
 - ≻Time dependent, quantitative data
- Digital Calipers
 - Precise final displacement



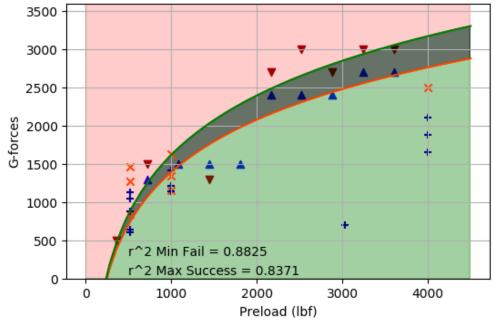
²⁰ Test Results



Silicon70 qin max final displacement: 1 (mm), max oscillation: 10 (mm)



Silicon30 qin max final displacement: 1 (mm), max oscillation: 10 (mm)



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Limitations & Future Research



Simulation Limitations

- More rotation in plates than observed in testing
- μ_s changes based on compression

Deterministic simulation of stochastic process

Advisor Limitations

- Maximum success point dependent on range swept
- Oscillations do not always terminate in time
- Cannot predict specific displacements

Test Limitations

- Pulsed shock duration is inconsistent
- Bolts lose compression after shock
- Shock amplitude is inconsistent and infeasible to predict a priori









- ²⁵ Acknowledgments
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 Jesse Powers
 Mentors
 Greg Neugebauer
 - >Wesley Greenwood
 - ≻Neal Hubbard
 - ≻Ramon Reyes

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²⁶ References

[1] Weber, B., Scheibert, J., de Boer, M.P. *et al.* Experimental insights into adhesion and friction between nominally dry rough surfaces. *MRS Bulletin* **47**, 1237–1246 (2022). https://doi.org/10.1557/s43577-022-00464-6

[2] Kogut, L., and Etsion, I. (January 13, 2004). "A Static Friction Model for Elastic-Plastic Contacting Rough Surfaces ." ASME. *J. Tribol.* January 2004; 126(1): 34–40. https://doi.org/10.1115/1.1609488

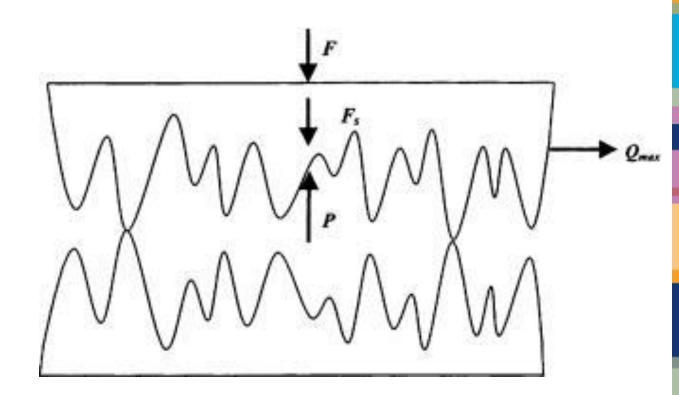
[3] Yang, J., and Komvopoulos, K. (April 7, 2005). "A Mechanics Approach to Static Friction of Elastic–Plastic Fractal Surfaces ." ASME. *J. Tribol.* April 2005; 127(2): 315–324. https://doi.org/10.1115/1.1828080

[4] Risch, Brian & Fallahmohammadi, Ehsan & Anderson, Nick. (2018). Structurally and Environmentally Robust Flexible Ribbon for High Fiber Density Cables Compatible with Mass Fusion Splicing.

- ²⁷ Friction Modeling With Rigid Materials
 - Dependent on micro and nano scale surface features

True surface area changes friction coefficient

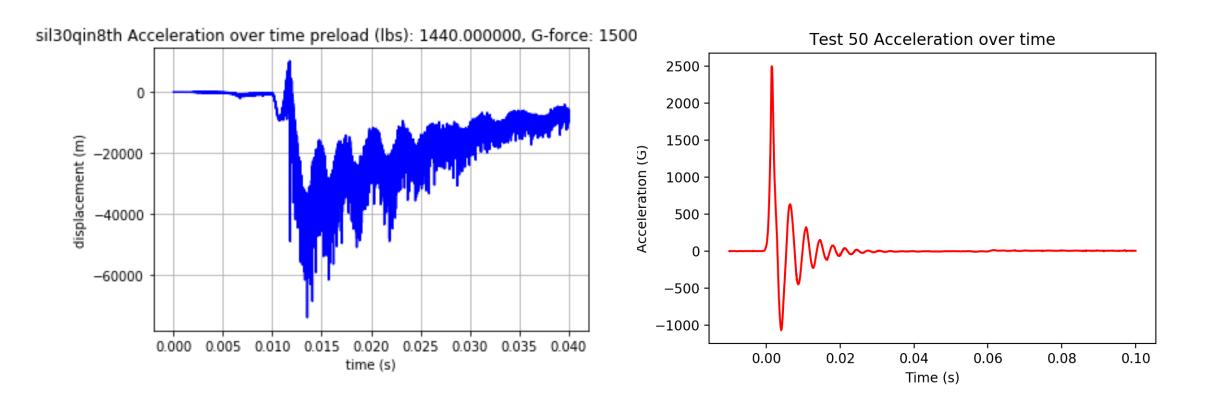
Compression independent



FEA often required for simulation

[2]

²⁸ Simulation Acceleration VS Test Acceleration



Design Considerations & Central Questions

- > Will a shock displace a compressed part?
- > What are the failure conditions?
- > How much compression is there?
- > How large is the shock?

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- How long is the shock?
- > What is the pad material?
- > What is the pad geometry?

